

N F P A

Fluid Power

VEHICLE

Challenge



NFPA
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Foundation

FINAL PRESENTATION
UNIVERSITY OF UTAH
FACULTY ADVISOR: DR. CHEN
04/11/2019



Department of
MECHANICAL ENGINEERING
THE UNIVERSITY OF UTAH

Photo of Vehicle

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Team Introduction



Dr. Chen



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Problem Statement



- Create a vehicle that is human powered, and uses fluid power components to transmit power from user to forward motion.
- To compete in the Fluid Power Vehicle Challenge.
- Gain experience in fluid power systems.
- Create networking opportunity with industry experts.
- Establish a recurring project for the University of Utah Mechanical Engineering Department.

Summary of Midway Review



- Our design objectives were:
 1. Low cost
 2. Reliability
 3. Efficient
 4. User friendly
 5. Meet/exceed competition specifications
- The vehicle was designed around the individual fluid circuit components from our circuit diagram.
 - Vehicle Design had a focus of simplicity.
- Fluid Circuit Design
 - Goal was to minimize the number of valves with a greater emphasis on individual components being able to perform in multiple drive modes.

Summary of Midway Review



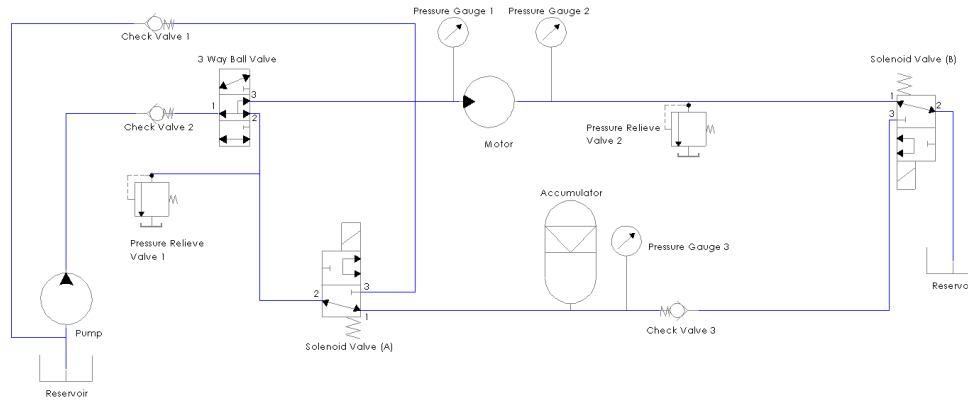
- Hardware Selection from Analysis
 - We selected a .58 CID pump, .62 CID bidirectional motor, and sized our lines based on advice (-6 for pressure/return lines, -12 for the suction line).

Basic Torque Calculations		Equation	Value	Units
Rear Wheel Torque Necessary to Create movement	$T_r = \text{Frictional coefficient} * \text{rear wheel weight} * \text{wheel radius}$		116	[ft-lb]
Front Wheel Torque Necessary to Create movement	$T_f = \text{Frictional coefficient} * \text{front wheel weight} * \text{wheel radius}$		97	[ft-lb]
Fluid motor torque	$\text{Torque} = (\text{PSI} * d) / (100 * .0628)$		247	[in-lb]
Torque conversion from [in-lb] to [ft-lb]	$[\text{ft-lb}] = [\text{in-lb}] / 12$		21	[ft-lb]

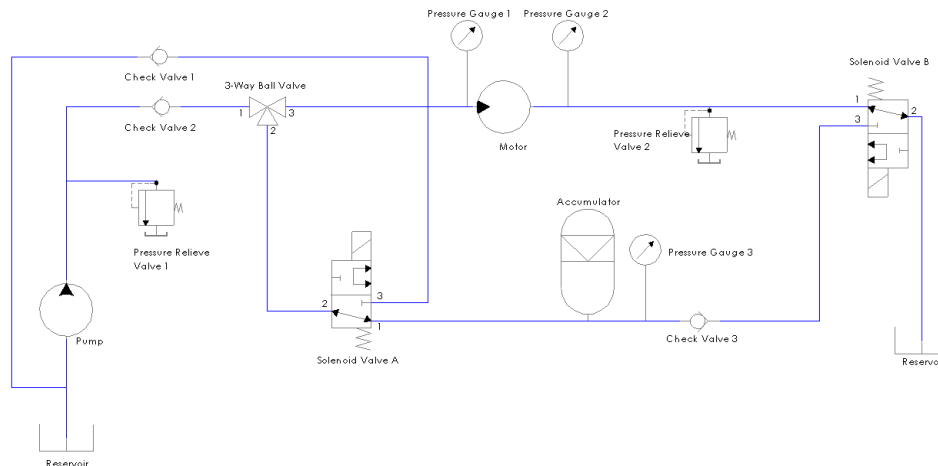
Pump Calculations		Equation	Value	Units
Flow out of pump	$\text{Flow out} = (\text{Fluid displaced by motor per revolution} * \text{pump RPM}) / 231$		3	[GPM]
Flow into pump	$\text{Flow in} = \text{Flow out} / \text{Volumetric efficiency}$		3.3	[GPM]
Overall efficiency	$\text{Overall efficiency} = \text{volumetric efficiency} * \text{mechanical efficiency}$		0.89	[]
Input horsepower	$\text{Input horsepower} = (\text{Pump flow rate out [GPM]} * \text{pressure [psi]}) * .0007$		6	[hp]
Input power	$\text{Input Power} = \text{input horsepower} * 745.7$		4718	[W]
Gear ratio needed for pump	$\text{Gear ratio} = \text{Pump RPM} / \text{input RPM}$		15	[]

Wheel Calculations		Q=GPM, d=CID		
Fluid motor speed	$\text{Speed} = (231 * Q / d)$	1123	[RPM]	
Gear ratio for wheel	$\text{Torque needed} / \text{Torque we have}$	4.7	[]	
RPM of wheel	$\text{Motor Speed} / \text{Gear ratio (rounded up)}$	160.4	[RPM]	
Speed of the bike	$(\text{RPM} * 2 * \pi * \text{radius}) / 60$	18.2	[feet/sec]	
		12	[MPH]	

Summary of Midway Review



Fluid Circuit Diagram Before Midterm Review

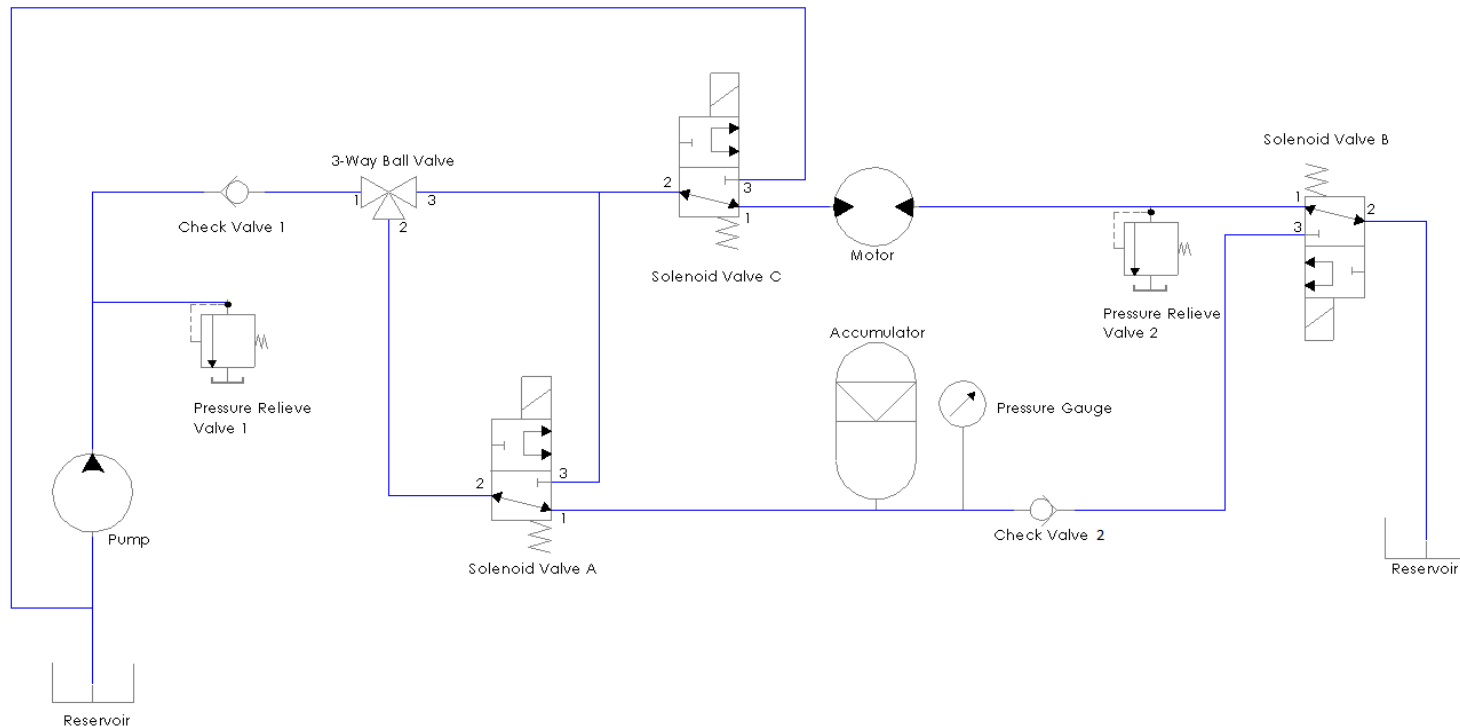


Fluid Circuit Diagram After Midterm Review

Final Fluid Circuit Diagram



1. Consists of 4 different modes:
 - Direct Drive
 - Charging
 - Discharging
 - Regenerative Braking
2. Control by valves positioning using 2 switches and 1 handle.
3. Safety- Pressure relief valves.



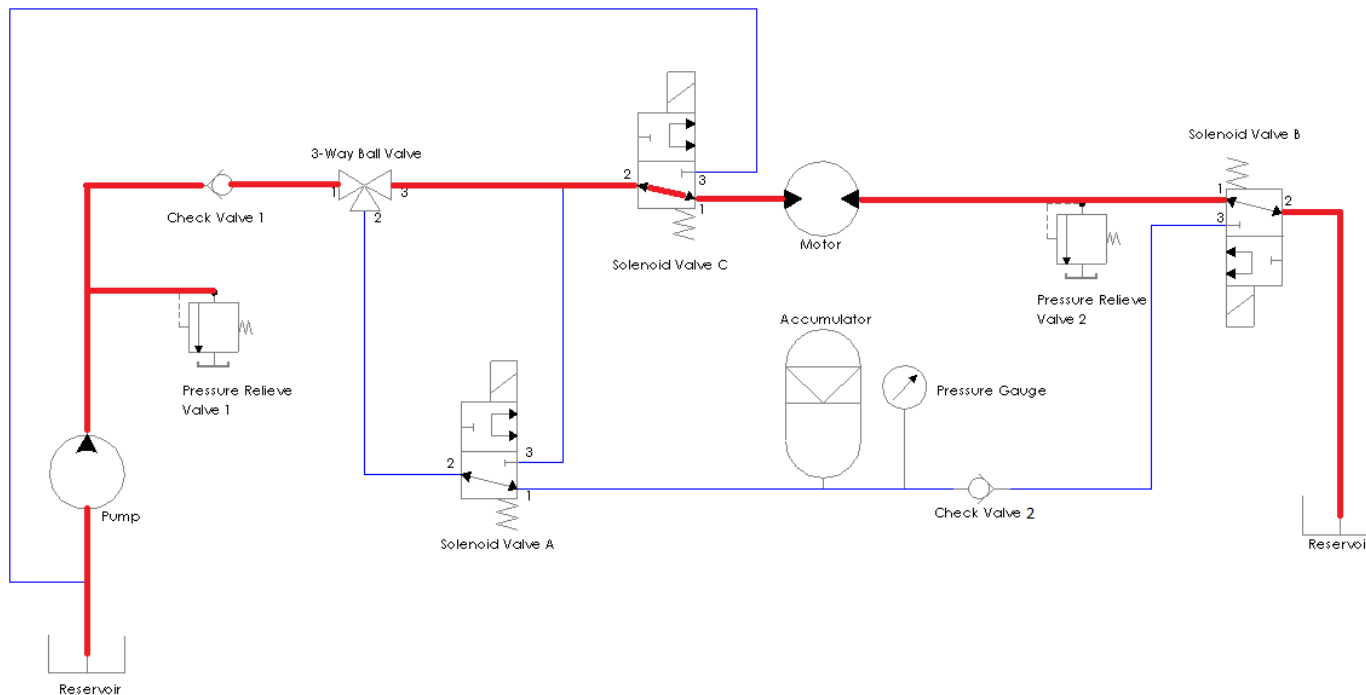
Direct Drive Mode

Task: Pumping the fluid directly from reservoir to the motor

Challenge: Sprint Race

Valves Position:

- 3 way ball valve = 90° Angle (allow flow from pump to motor)
- Solenoid Valve (A) = de-energized
- Solenoid Valve (B) = de-energized (allow flow from 2 to 1)
- Solenoid Valve (C) = de-energized (allow flow from 1 to 2)



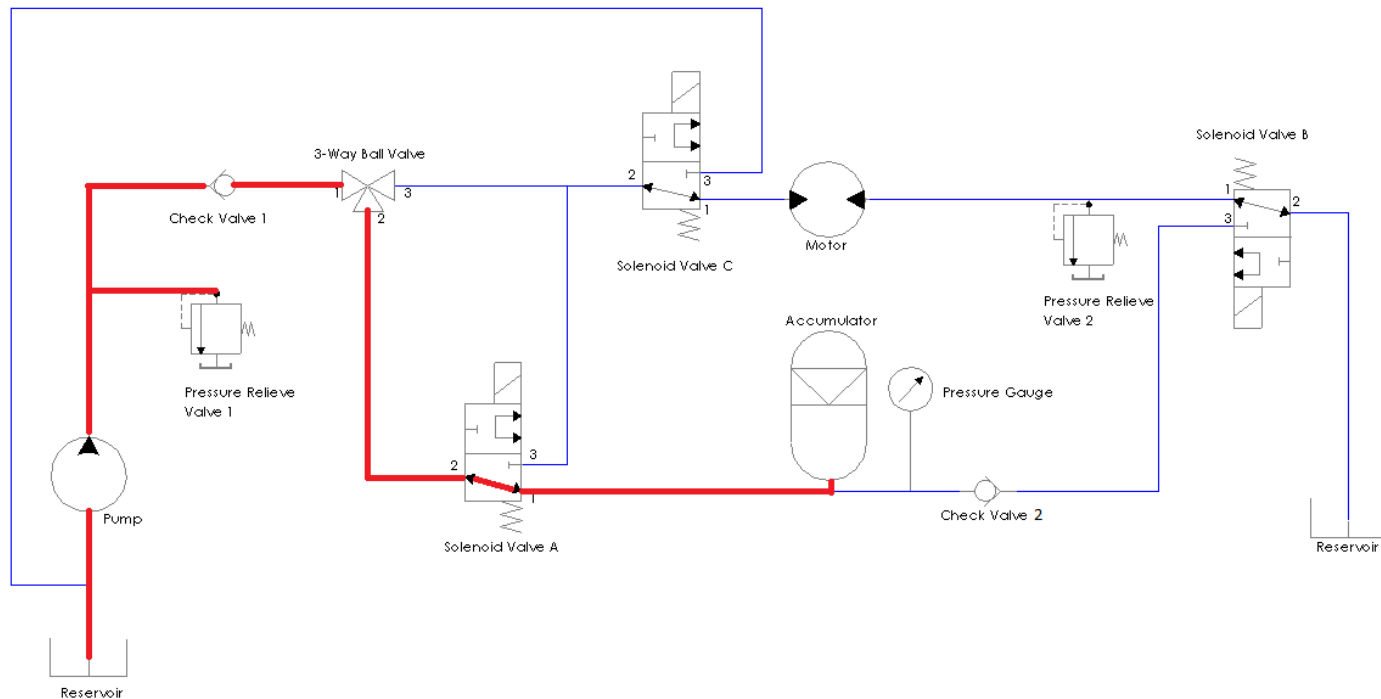
Charging Mode

Task: Charging the accumulator

Challenge: Efficiency test

Valves Position:

- 3 way ball valve = 0° Angle (allow flow from pump to accumulator)
- Solenoid Valve (A) = de-energized (allow flow from 2 to 1)
- Solenoid Valve (B) = de-energized
- Solenoid Valve (C) = de-energized



Regenerative Braking Mode

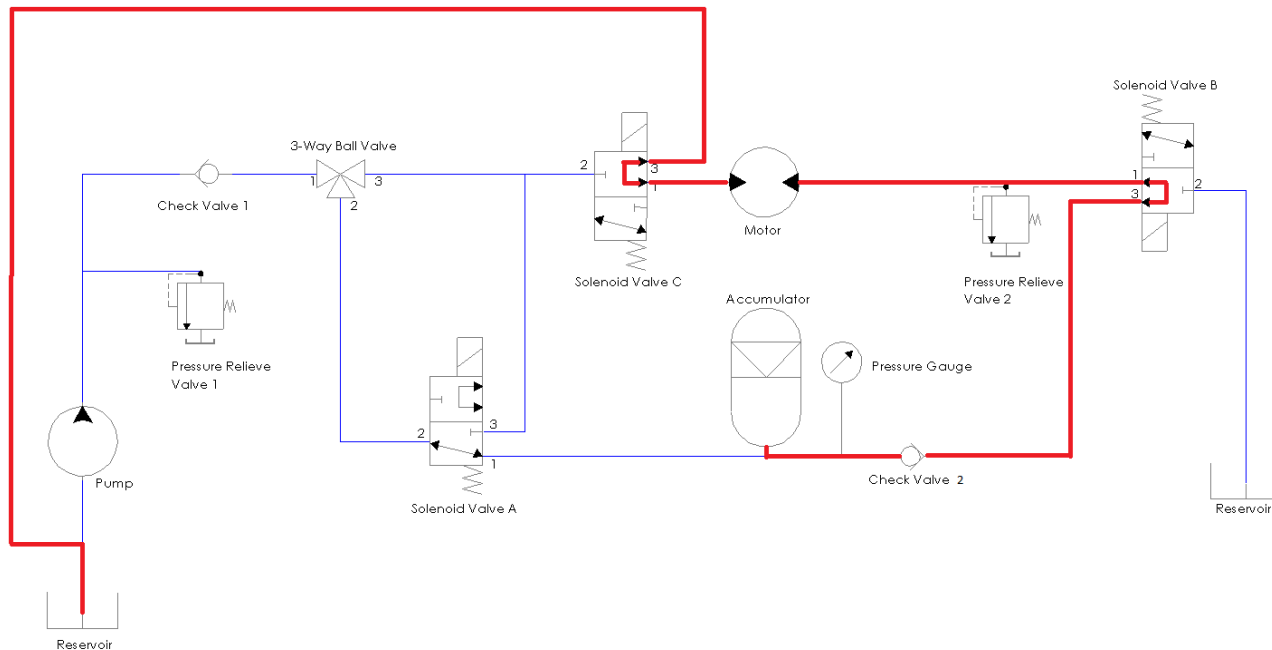


Task: Use the momentum from the bike to turn the motor to pressurize fluid back into accumulator

Challenge: Endurance test

Valves Position:

- 3 way ball valve = any
- Solenoid Valve (A) = de-energized
- Solenoid Valve (B) = energized (allow flow from 1 to 3)
- Solenoid Valve (C) = energized (allow flow from 3 to 1)



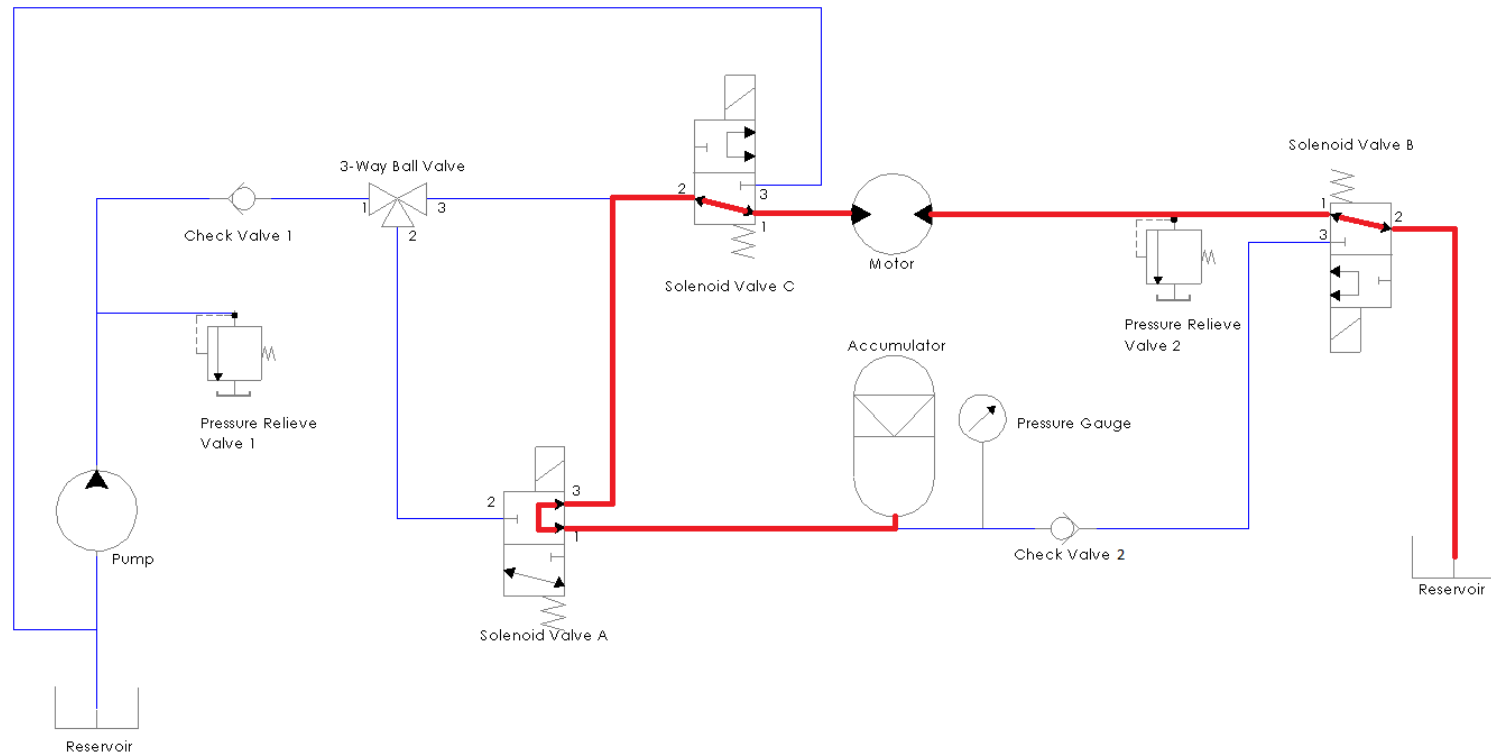
Discharging Mode

Task: Discharge the accumulator

Challenge: Efficiency and endurance test

Valves Position:

- 3 way ball valve = Any
- Solenoid Valve (A) = energized (allow flow from 1 to 3)
- Solenoid Valve (B) = de-energized
- Solenoid Valve (C) = de-energized

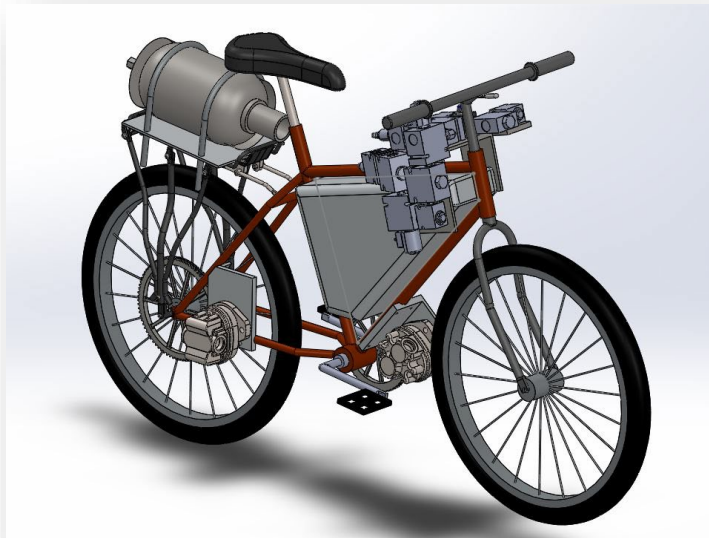


Vehicle Construction



- Initial Design and layout were modeled in Solidworks, in order to help with fitment and design of bracketry and layout of vehicle
- From this model, work was divided up by systems
 - Pump and Motor
 - Valve Assembly
 - Electrical Circuitry
 - Accumulator
 - Reservoir

SolidWorks Model



Final Fluid Power Vehicle



Vehicle Construction



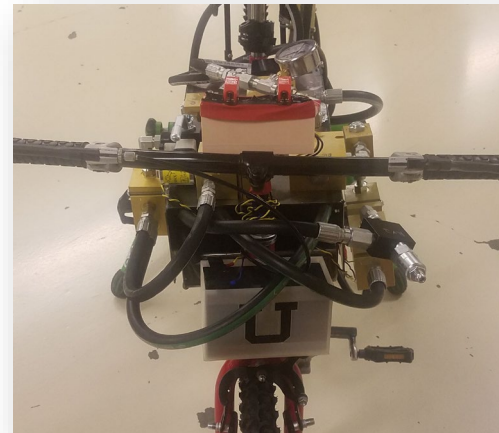
- Pump and Pump Gearing
 - Designed to fit under bottom tube of bike, the pump is turned by a chain attached to a gear on the pedals
 - Due to interference and chain binding issues, the intermediate gear was removed, lowering the overall gear ratio
- Motor and Motor Gearing
 - The motor was designed to mount on the back tire support frame
 - The gear was bolted directly to the hub of the wheel
- Accumulator
 - The Accumulator was mounted on a bicycle rack behind the rider



Vehicle Construction



- Valve Assembly
 - Due to size of connection pieces, the final valve assembly was broken up into parts, allowing for different bodies to be mounted in different locations
- Reservoir
 - Based off of the design in the SolidWorks model, the reservoir was welded out of steel plates. Several port connections were cut and welded into the reservoir body.
- Electrical Circuitry
 - An Arduino controlling the solenoid switches was mounted on the handlebars, with the battery situated in front of the handlebars



Vehicle Testing



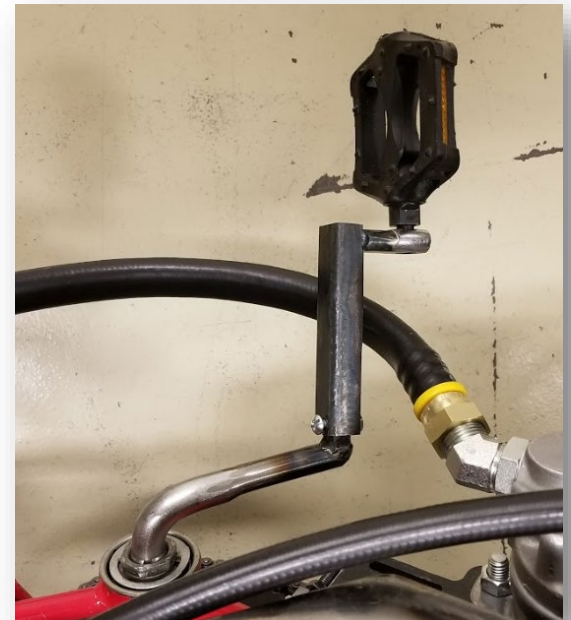
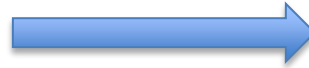
Direct Drive Mode

- Gear Ratio = 7:1

Test 1

Parameter	Value
Distance Traveled	270 ft
Time Elapsed	60 seconds
Average Speed	3 mph

Identified loose pedal extensions causing difficult pedaling



Vehicle Testing

Direct Drive Mode

- Gear Ratio = 7:1



Test 2

Secured pedal extensions
to remove play



Parameter	Value
Distance Traveled	120 ft
Time Elapsed	16 seconds
Average Speed	5 mph
Avg Speed from Test 1	3 mph
Improvement	2 mph

Vehicle Testing



Accumulator Drive Distance Analysis:

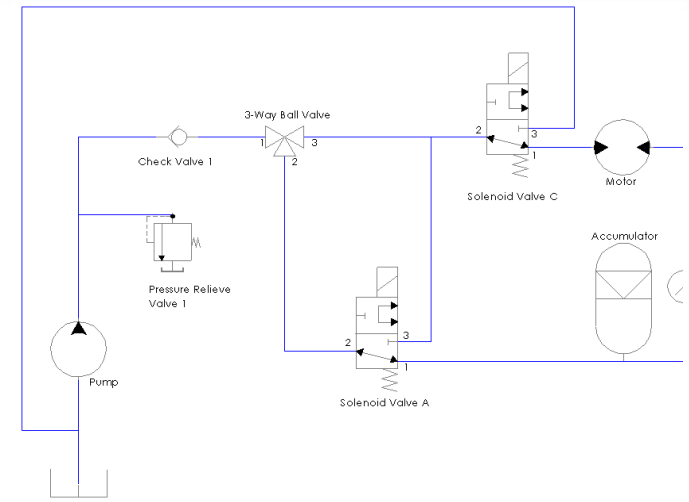
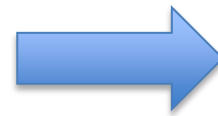
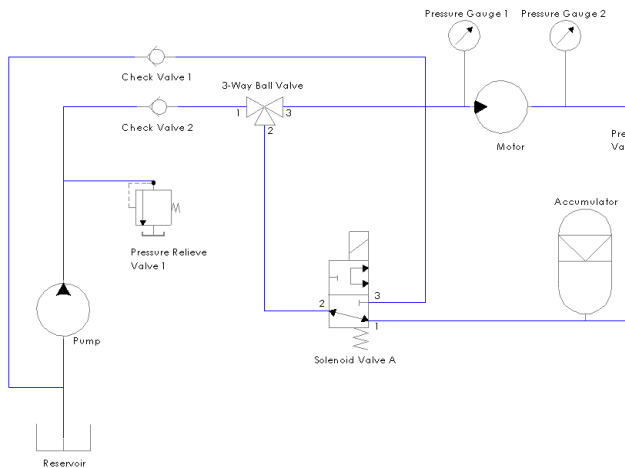
Parameter	Value
Pre-charged Pressure	900 psi
Fluid Pressure	1100 psi
Distance Traveled	100 ft



Vehicle Testing

Regenerative Braking

- In our initial test, the regenerative braking didn't work due to difficulty of overcoming the check valve in the circuit
- The Check valve was replaced by a Solenoid valve



Lessons Learned



- A manifold would reduce the number of fittings and valve bodies required
- Suction lines should be short and direct, and not have a check valve on low pressure side
- Can save money on suction hoses by not buying 3000 psi rated hoses
- Beware of current draw through a transistor for solenoid control



**Thank you to the NFPA and
industry sponsors!**